Vibration Analyst Category I Equations

FORCES

Mass Unbalance

$$F = Me \left(\frac{2\pi N}{60}\right)^2$$

M = W/g

W = weight of rotor or balance weight, lb

e = rotor eccentricity or radius of balance weight, in

 $g = gravitational constant, 386.1 in/s^2$

N = RPM

Spring Force

F = Kx

K = stiffness of spring, lb/in

x = relative deflection, in

Damping Force

 $F = C \, \dot{x}$

C = damping constant, lb-s/in

 \dot{x} = relative velocity

Inertia Force

 $F = M \ddot{x}$

 $M = mass, lb-s^2/in$

 $\ddot{\mathbf{x}}$ = acceleration, in/s²

MOTIONS

Velocity (in/s)

 $V = D(2\pi f)$

D = peak displacement, in

f = frequency, cycles/s (CPS)

 $\pi = 3.14$

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Acceleration

$$A = V(2\pi f)$$

A = acceleration,
$$in/s^2$$

1 g = 386.1 in/s^2

FREQUENCIES

Bearing Frequencies

$$\begin{aligned} &\mathsf{FTF} = \left(\frac{\Omega}{2}\right) \! \! \left[1 - \! \left(\frac{\mathsf{B}}{\mathsf{P}}\right) \! \mathsf{cos} \, \mathsf{CA} \right] \\ &\mathsf{BPFI} = \! \left(\frac{\mathsf{N}}{2}\right) \! \Omega \! \! \left[1 + \! \left(\frac{\mathsf{B}}{\mathsf{P}}\right) \! \mathsf{cos} \, \mathsf{CA} \right] \\ &\mathsf{BPFO} = \frac{\mathsf{N}}{2} \Omega \! \! \left[1 - \! \left(\frac{\mathsf{B}}{\mathsf{P}}\right) \! \mathsf{cos} \, \mathsf{CA} \right] \\ &\mathsf{BSF} = \! \left(\frac{\mathsf{P}}{2\mathsf{B}}\right) \! \Omega \! \! \left[1 - \! \left(\frac{\mathsf{B}}{\mathsf{P}}\right)^{\! 2} \mathsf{cos}^{2} \, \mathsf{CA} \right] \end{aligned}$$

FTF = fundamental train frequency

BPFI = ball pass frequency, inner race

BPFO = ball pass frequency, outer race

BSF = ball spin frequency

RPM = shaft speed

CA = contact angle

 Ω = machine speed

N = number of rolling elements

P = pitch diameter, in

B = ball or roller diameter, in

Bearing defect frequencies are same units as machine speed

General Guideline Bearing Frequencies

(for use in FMax selection ONLY)

 $BPFO = 0.41 \times RPM \times N$

 $BPFI = 0.59 \times RPM \times N$

 $FTF = 0.41 \times RPM$

 $BSF = 0.22 \times RPM \times N$

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Natural Frequency

$$f_n = \frac{1}{2\pi} \sqrt{\frac{k}{m}}$$

k = stiffness, lb/in

m = w/g

w = weight, lb

 $g = gravitational constant, 386.1 in/s^2$

 f_n = natural frequency of a single-degree-of-freedom system, Hz

Roll Frequency

$$f = \frac{V}{5\pi D}$$

V = web velocity, ft/min

D = roll diameter, in

f = frequency, Hz

SIGNAL PROCESSING

Dynamic Range

$$dB = 20 log \frac{V_m}{V_r}$$

$$\frac{V_m}{V_r} = 10^{\frac{dB}{20}}$$

 $V_{\rm m}$ = voltage measured

 V_r = voltage reference

dB = decibels

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RMS

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peak = 1.414 \text{ rms}
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Resolution

Resolution = (frequency span x window noise factor x 2)/#FFT lines

window noise factor =

- 1.0 for uniform window
- 1.5 for Hanning window
- 3.8 for flat top window

Data Acquisition Time (DAT)

DAT = # FFT lines/frequency span

Default Frequency Spans

Operating Speed = $10 \times RPM$ Rolling Element Bearings = $10 \times BPFI$ Fluid Film Bearings = $10 \times RPM$

Vane/Blade Pass= 3 x # Vanes/Blades x RPMElectrical= 3 x 2X Line FrequencyGear Mesh= 3 x Gear Mesh Frequency

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